# POTENTIAL ENERGY SAVING WITH HIGH $\Delta T$ CHILLED WATER DESIGNS COMPARED TO THE CONVENTIONAL LOW $\Delta T$ DESIGN FOR OFFICE BUILDINGS

JOSEPH A. I.

208895L

Thesis/ Dissertation Submitted in Partial Fulfilment of the Requirements for the Master of Science in Building Services Engineering

Department of Mechanical Engineering

University of Moratuwa Sri Lanka

September 2023

DECLARATION OF THE CANDIDATE AND THE SUPERVISOR

I declare that this is my own work and this thesis/dissertation does not incorporate

without acknowledgement any material previously submitted for a Degree or Diploma

in any other University or institute of higher learning and to the best of my knowledge

and belief it does not contain any material previously published or written by another

person except where the acknowledgement is made in the text.

Also, I hereby grant to University of Moratuwa the non-exclusive right to reproduce

and distribute my thesis/dissertation, in whole or in part in print, electronic or other

medium. I retain the right to use this content in whole or part in future works (such as

articles or books).

Signature:

Date: 27/09/2023

The supervisor/s should certify the thesis/dissertation with the following declaration.

The above candidate has carried out research for the Masters/MPhil/PhD

thesis/dissertation under my supervision.

Signature of the supervisor:

Date: 07/10/2023

i

### **ABSTRACT**

This thesis investigates the potential energy saving benefits associated with high delta T chilled water designs compared to conventional low delta T designs in office buildings. This high delta T requirement specified in the American Society of Heating, Refrigerating and Air-Conditioning Engineers (ASHRAE) 90.1: 2016 under subsection 6.5.4.: "Chilled Water coil selection and theoretical background of temperature difference towards energy efficiency". The aim of the study is to evaluate the capital cost reduction and annual energy savings achievable through the implementation of high delta T configurations and finally identify the applicability in tropical countries. The study followed a comprehensive research methodology that involved analysing 03 chilled water delta T options and assessing their impact on capital costs and energy consumption.

The findings of the study demonstrate a significant cost savings and energy efficiencies associated with high  $\Delta T$  chilled water designs. A total capital cost reduction of 2.8% for the  $\Delta T$ =7°C system and 3.6% for the  $\Delta T$ =8.3°C system was observed compared to the conventional  $\Delta T$ =5°C system. Additionally, the annual energy analysis revealed a 0.97% reduction in energy costs for the  $\Delta T$ =7°C system and a higher energy cost saving of 1.93% for the  $\Delta T$ =8.3°C system compared to the  $\Delta T$ =5°C system.

These findings highlight the potential benefits of adopting high delta T chilled water designs in office buildings. The results not only demonstrate cost-effectiveness but also underscore the importance of energy efficiency and sustainability in building design and operation. The outcomes of this study contribute to the existing body of knowledge in the field and provide valuable insights for building professionals and stakeholders seeking optimal cooling system solutions. By embracing high delta T designs, practitioners can achieve tangible benefits in terms of reduced capital expenditures and ongoing energy consumption, leading to a more sustainable and economically viable built environment.

**Keywords:** Energy efficiency, high delta T, chilled water, office building, tropical country

### **ACKNOWLEDGEMENTS**

I would like to express my deep gratitude to my supervisor, Dr. Anusha Wijewardane, and co-supervisor, CEng. Suranga Lekamwasam, for their patient guidance, enthusiastic encouragement, and the necessary technical instructions they provided whenever required. Their invaluable support has been instrumental in the successful completion of this thesis.

I would also like to extend my heartfelt thanks to my colleagues, all academic staff, resource persons, and non-academic staff of the MSc program. Their insights and expertise have greatly assisted in shaping this study and broadening my understanding of the subject matter.

Furthermore, I would like to seize this opportunity to express my sincere appreciation to my family members, especially my loving wife, whose unwavering support has been instrumental throughout this journey. Their encouragement and understanding have played a vital role in enabling me to successfully complete this literature review.

I am deeply grateful to all those mentioned above for their contributions, guidance, and unwavering support, which have been invaluable in the completion of this research.

# TABLE OF CONTENTS

DECLARATION OF THE CANDIDATE AND THE SUPERVISOR	i
ABSTRACT	ii
ACKNOWLEDGEMENTS	iii
TABLE OF CONTENTS	iv
LIST OF FIGURES	. vii
LIST OF TABLES	viii
LIST OF NOMENCLATURE	X
CHAPTER 1	1
1. INTRODUCTION	1
1.1. Introduction/ Background	1
1.2. Introduction of Chapters	2
1.3. Aim	4
1.4. Objectives	4
1.5. Methodology	5
CHAPTER 2	6
2. HIGH DELTA T DESIGN	6
2.1. Introduction	6
2.1.1. Use of Primary Variable Chilled Water Pumping System	7
2.1.2. Reduce the Chiller Lift	7
2.1.3. Use VFD Condenser Water Pumps for Commissioning Purpose	8
2.1.4. Use of VFD Cooling Towers and Selecting Correct Cooling Tower Ty	pe9
2.1.5. Design of High Delta T Chilled Water System.	. 10
2.2. Explanation of Exceptions to the High Delta T	. 13
2.3 Delta T Recommendations Practice in the Industry	14

2.4.	Advantages and Disadvantages of High Delta T Systems	
2.5.	Approaches on High Delta-T Systems by the Other Countries	
2.5	5.1. High Delta T Application in Singapore Conditions	
2.5	5.2. Achieving High Chilled Water Delta T - An Indian Study	
2.5	5.3. High Delta T Application in Korea	
2.6.	Summary of Literature Review	
2.7.	Research Gap21	
CHAPT	TER 3	
3. M	ETHODOLOGY22	
3.1.	Theoretical Analysis of the Impact of Increased Chilled Water Delta T 22	
3.1	1.1. Effect on the Water Side Equipment	
3.1	1.2. Effect on the Air Side Equipment	
3.2.	Introduction to the Selected Design	
3.3.	Cooling Load Simulation	
3.3	3.1. Indoor Design Conditions	
3.3	3.2. Weather and Outdoor Parameters	
3.3	3.3. Design Temperatures	
3.3	3.4. Chilled Water System Design Conditions	
3.4.	Selection of Air Handling Unit (AHU)	
3.4	4.1. Selection of the AHU Coil	
3.4	4.2. Input Data to the Selection Software	
3.5.	Evaluation of the Bill of Quantity (BOQ)	
3.5	5.1. BOQ Items in Chilled Water System	
3.5	5.2. Price of Chiller for Different Delta T Conditions	
3.5	5.3. Cost of Chilled Water Piping System for Different Delta T Conditions . 39	
3 4	5.4 Price of Chilled Water Pump for Different Delta T Conditions 45	

•	3.5.5. Price of Air Handling Unit for Different Delta T Conditions	45
СНА	PTER 4	49
4.	RESULTS	49
4.1	. Results of the Cooling Load Simulation	49
4.2	Sizing of Chiller Plant Equipment	52
4.3	Chiller Plant Simulation	54
4.4	Air Side Equipment Selection	60
4	4.4.1. Effect on AHU Dimensions and Weight	62
2	4.4.2. Effect on AHU Power Consumption	64
4.5	Comparison of the Cost Impact	72
4	4.5.1. Cost Evaluation of System	73
4	4.5.2. Evaluation of Annual Energy Cost	76
4.6	6. Quantification of the Energy Saving Based on Total Cooling Load	76
4.7	Comparison of Results with Other Studies	77
4	4.7.1. Results of Research Conducted in Singapore	77
4	4.7.2. Results of Research Conducted in India	77
4	4.7.3. Results of Research Conducted in Korea	78
4	4.7.4. Comparison of the Results	78
СНА	PTER 5	79
5.	DISCUSSION AND CONCLUSION	79
5.1	. Discussion	79
5.2	Conclusion.	80
REFE	ERENCES	82

# LIST OF FIGURES

Figure 1 : Chilled water system diagram [3]
Figure 2 : Reduce chiller lift by increased LCHWT
Figure 3 : Reduce chiller lift by reduced ECWT
Figure 4 : Chilled water-cooling coil
Figure 5 : Conventional vs high delta T systems at full load [11]
Figure 6 : Sectional view of the proposed office building
Figure 7 : Typical office floor
Figure 8 : Schematic diagram of the chilled water system
Figure 9 : Chilled water system layout for chiller room and pump room41
Figure 10 : Hourly cooling load profile of the plant
Figure 11 : Monthly energy usage data of chilled water pumps
Figure 12 : Monthly energy usage data of chiller
Figure 13 : Annual chiller plant room energy consumption
Figure 14 : Chiller efficiency pattern for different chilled water temperatures 59
Figure 15 : Change of volume of AHU for different chilled water configurations $\dots 63$
Figure 16 : Change of weight of AHU for different chilled water configurations 64
Figure 17 : Change of coil area of AHU for different chilled water configurations 66
Figure 18: Change of absorbed power of AHU for different chilled water
configurations67
Figure 19 : Change of annual energy consumption of AHU vs delta T70
Figure 20 : Annual chilled water system energy consumption

# LIST OF TABLES

Table 1 : Recommended high delta T values in the industry	14
Table 2 : Considered case scenarios relevant to tropical climates	20
Table 3 : Building construction details	27
Table 4: Monthly Minimum and Maximum Temperatures from Software Data	Base
	29
Table 5 : Air Handling Unit/ Fan Coil Unit (FCU) selection of the room	30
Table 6 : Coil configuration and construction options	32
Table 7 : Constant coil parameters and values	33
Table 8: BOQ items and affect to the increased delta T	36
Table 9 : Price proposal for magnetic bearing chiller	38
Table 10 : BOQ for Chilled Water Pipe Network - delta T= 5°C	42
Table 11 : BOQ for Chilled Water Pipe Network - delta T= 7°C	43
Table 12 : BOQ for Chilled Water Pipe Network – delta T= 8.3°C	44
Table 13 : Price proposal for chilled water pumps	45
Table 14 : Price proposal for air handling units selected – delta T= 5°C	46
Table 15 : Price proposal for air handling units selected – delta T= 7°C	47
Table 16 : Price proposal for air handling units selected – delta T= 8.3°C	48
Table 17: Maximum cooling capacity of air-conditioned spaces	49
Table 18: Plant room cooling capacity	51
Table 19: Chiller and pump energy consumption results	54
Table 20 : Annual energy usage of the chiller plant	57
Table 21: Efficiency Comparison based on different chilled water temperatures.	59
Table 22 : AHU selection main input data summary	60
Table 23 : Details of the Selected AHUs	61
Table 24 : AHU dimension details	62
Table 25 : AHU air side parameters	65
Table 26: AHU water side parameters	67
Table 27 : Annual energy usage of AHUs	69
Table 28: Annual energy usage of the chilled water system	70
Table 29: Price proposal for chilled water system when delta $T = 5^{\circ}C$	73

Table 30 : Price proposal for chilled water system when delta $T=7^{\circ}C$	73
Table 31 : Price proposal for chilled water system when delta $T = 8.3$ °C	74
Table 32 : Evaluation of capital cost compared to 5°C dT system	75
Table 33 : Summary of estimated energy cost of each system	76
Table 34: Quantification of annual energy saving per refrigerant tonnage	77

### LIST OF NOMENCLATURE

**Abbreviation Description** 

ACH Air Changes per Hour

ASHRAE American Society of Heating, Refrigerating and Air

**Conditioning Engineers** 

BOQ Bill of Quantity

BS British Standards

CAV Constant Air Volume

CDWR Condenser Water Return

CDWS Condenser Water Supply

CHWR Chilled Water Return

CHWS Chilled Water Supply

CIBSE Chartered Institution of Building Services Engineers

COP Coefficient of Performance

DB Dry Bulb

Delta T/ ΔT Temperature Difference

ECWT Entering Condenser Water Temperature

ESP External Static Pressure

FA Fresh Air

GHG Greenhouse Gas

IAQ Indoor Air Quality

LCHWT Leaving Chilled Water Temperature

RA Return Air SA Supply Air

SFP Specific Fan Power

SLSEA Sri Lanka Sustainable Energy Authority

SS Single Split

TCC Total Cooling Capacity

VRV/ VRF Variable Refrigerant Volume/ Flow

WB Wet Bulb

## **Notations Description**

 $\begin{array}{ll} Q & & \text{Heat transfer rate} \\ \rho_w & & \text{Density of water} \\ V_w & & \text{Water flow rate} \end{array}$ 

Cpw Specific heat of water

 $\Delta T_{\rm w}$  Chilled water temperature difference

P<sub>P</sub> Pump power

 $\Delta P_{\rm w}$  Pressure drop of the circuit

 $\begin{array}{ll} \eta_{p} & & Pump \ efficiency \\ \rho_{a} & & Density \ of \ air \\ V_{a} & & Air \ flow \ rate \end{array}$ 

Cpa Specific heat of air

MLTD<sub>a</sub> Air side mean logarithmic temperature difference MLTD<sub>w</sub> Water side mean logarithmic temperature difference

P<sub>F</sub> Fan power

 $\begin{array}{ll} \Delta P_{coil} & Coil \ pressure \ loss \\ \eta_F & Fan \ efficiency \end{array}$ 

# **Subscripts** Description

a Air
F Fan
P Pump
w Water