

MODELLING OF COMBUSTION IN A SINGLE-BURNER BIOGAS FIRED **COOKING STOVE**



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DECLARATION

"I certify that this thesis does not incorporate, without acknowledge, any material previously submitted for a degree or diploma in any university or higher educational institution in Sri Lanka or abroad and to the best of my knowledge and belief it does not contain any material previously published or written by another person except where due reference is made in the text".

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ABSTRACT

A variety of stoves are used for household cooking in Sri Lanka. Fuel-wood, Liquefied Petroleum Gas (LPG), Electricity, Kerosene oil, Biogas etc. are the common cooking fuels used. Combustion process in a cooking stove is a complicated phenomenon. It is very difficult to predict the distributions of temperature, flow properties and combustion product concentrations of the cooking stove. It is emphasized that a detailed understanding of the combustion process taking place in a cooking stove is essential for the development of better stove designs. Computational modelling is an efficient tool that could be used successfully in describing the combustion in cooking stoves. Modelling of combustion in a cooking stove that uses a gaseous fuel is comparatively easier than that uses a solid fuel, mainly due to the complexity of the combustion process that the solid fuel undergoes. On this basis present work is involved in the modelling of combustion taking place in a biogas fired cooking stove using SOFIE, a Computational Fluid Dynamics (CFD) code extensively used for fire modelling. The combustion flow field of the stove has been modelled using the k-E turbulence model for turbulence and one-step reaction fast chemistry represents combustion chemistry. Simulations are conducted for the biogas cooking flame alone and also for the single-burner biogas fired stove with a squareshaped cooking pan. Temperature, density and combustion product concentration predictions have been made using simulations. The predicted temperatures are compared with the experimental measurements. The results generated could be used as a basis for further research in combustion in cooking stoves in order to develop better designs.

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NOMENCLATURE

 D_{f} Fin diameter (m) Acceleration due to gravity (ms⁻²) g Heat transfer coefficient (Wm⁻²K⁻¹) h Turbulent kinetic energy (m²s⁻²) k Thermal conductivity of fin material (Wm⁻¹K⁻¹) k_f L Length of fin (m) Time averaged mass fraction of fuel m_{6i} Time averaged mass fraction of oxidant m_{ox} Thermodynamic pressure (Nm⁻²) p Rate of conduction heat loss from a long fin (W) q_f Wall heat flux (Wm⁻²) q_w T_e Fin-end temperature (K) T_{f} Flame temperature (K) T_{o} Base temperature of fin (K) T_{p} Temperature at near wall point (K) T_s Sensor temperature (K) T_{w} Wall temperature (K) Ambient air temperature (K) T Time (s) U Velocity (ms⁻¹) Mass fraction of species a Y_{α}

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Greek Symbols

Dissipation of the turbulent kinetic energy (m²s⁻³) 3 emissivity of the thermocouple junction surface ϵ_{th} Polar angle (rad.) θ Von Karman constant κ Mixture molecular viscosity (Nsm⁻²) μ Turbulent eddy viscosity (Nsm⁻²) μ_t Kinematic viscosity (m²s⁻¹) ν Density of the mixture (kgm⁻³) ρ Stefan-Boltzmann constant (Wm⁻²K⁻⁴) σ Turbulent Prandtl/Schmidt number σ_{t} Characteristic turbulent time scale (s) τ Spectral transmissivity τ_{v} Wall shear stress (Nm⁻²) $\tau_{\mathbf{w}}$ Φ Azimuthal angle (rad.)